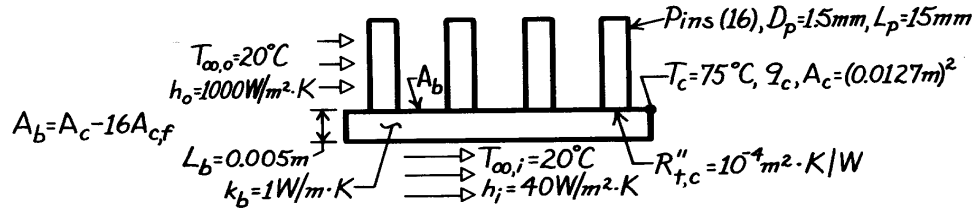


### PROBLEM 3.146

**KNOWN:** Geometry and cooling arrangement for a chip-circuit board arrangement. Maximum chip temperature.

**FIND:** (a) Equivalent thermal circuit, (b) Maximum chip heat rate.

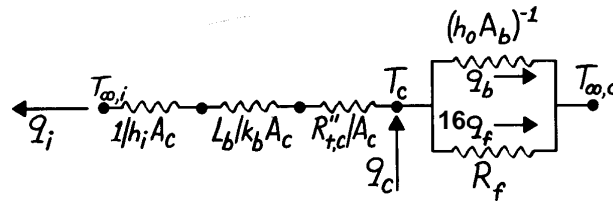
**SCHEMATIC:**



**ASSUMPTIONS:** (1) Steady-state conditions, (2) One-dimensional heat transfer in chip-board assembly, (3) Negligible pin-chip contact resistance, (4) Constant properties, (5) Negligible chip thermal resistance, (6) Uniform chip temperature.

**PROPERTIES:** Table A.1, Copper (300 K):  $k \approx 400$  W/m·K.

**ANALYSIS:** (a) The thermal circuit is



$$R_f = \frac{\theta_b}{16q_f} = \frac{\cosh mL + (h_o / mk) \sinh mL}{16(h_o P k A_{c,f})^{1/2} [\sinh mL + (h_o / mk) \cosh mL]}$$

(b) The maximum chip heat rate is

$$q_c = 16q_f + q_b + q_i.$$

Evaluate these parameters

$$m = \left( \frac{h_o P}{k A_{c,f}} \right)^{1/2} = \left( \frac{4h_o}{k D_p} \right)^{1/2} = \left( \frac{4 \times 1000 \text{ W/m}^2 \cdot \text{K}}{400 \text{ W/m} \cdot \text{K} \times 0.0015 \text{ m}} \right)^{1/2} = 81.7 \text{ m}^{-1}$$

$$mL = (81.7 \text{ m}^{-1} \times 0.015 \text{ m}) = 1.23, \quad \sinh mL = 1.57, \quad \cosh mL = 1.86$$

$$(h_o / mk) = \frac{1000 \text{ W/m}^2 \cdot \text{K}}{81.7 \text{ m}^{-1} \times 400 \text{ W/m} \cdot \text{K}} = 0.0306$$

$$M = \left( h_o \pi D_p k \pi D_p^2 / 4 \right)^{1/2} \theta_b$$

$$M = \left[ 1000 \text{ W/m}^2 \cdot \text{K} \left( \pi^2 / 4 \right) (0.0015 \text{ m})^3 400 \text{ W/m} \cdot \text{K} \right]^{1/2} (55^\circ \text{C}) = 3.17 \text{ W}.$$

Continued ...

### PROBLEM 3.146 (Cont.)

The fin heat rate is

$$q_f = M \frac{\sinh mL + (h_o/mk) \cosh mL}{\cosh mL + (h_o/mk) \sinh mL} = 3.17 \text{ W} \frac{1.57 + 0.0306 \times 1.86}{1.86 + 0.0306 \times 1.57}$$

$$q_f = 2.703 \text{ W}.$$

The heat rate from the chip top by convection is

$$q_b = h_o A_b \theta_b = 1000 \text{ W/m}^2 \cdot \text{K} \left[ (0.0127 \text{ m})^2 - (16\pi/4)(0.0015 \text{ m})^2 \right] 55^\circ \text{C}$$

$$q_b = 7.32 \text{ W}.$$

The convection heat rate from the board is

$$q_i = \frac{T_c - T_{\infty,i}}{(1/h_i + R''_{t,c} + L_b/k_b)(1/A_c)} = \frac{(0.0127 \text{ m})^2 (55^\circ \text{C})}{(1/40 + 10^{-4} + 0.005/1) \text{ m}^2 \cdot \text{K/W}}$$

$$q_i = 0.29 \text{ W}.$$

Hence, the maximum chip heat rate is

$$q_c = [16(2.703) + 7.32 + 0.29] \text{ W} = [43.25 + 7.32 + 0.29] \text{ W}$$

$$q_c = 50.9 \text{ W}.$$

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**COMMENTS:** (1) The fins are extremely effective in enhancing heat transfer from the chip (assuming negligible contact resistance). Their effectiveness is  $\varepsilon = q_f / \left( \pi D_p^2 / 4 \right) h_o \theta_b = 2.703 \text{ W} / 0.097 \text{ W} = 27.8$

(2) Without the fins,  $q_c = 1000 \text{ W/m}^2 \cdot \text{K} (0.0127 \text{ m})^2 55^\circ \text{C} + 0.29 \text{ W} = 9.16 \text{ W}$ . Hence the fins provide for a  $(50.9 \text{ W} / 9.16 \text{ W}) \times 100\% = 555\%$  enhancement of heat transfer.

(3) With the fins, the chip heat flux is  $50.9 \text{ W} / (0.0127 \text{ m})^2$  or  $q_c'' = 3.16 \times 10^5 \text{ W/m}^2 = 31.6 \text{ W/cm}^2$ .

(4) If the infinite fin approximation is made,  $q_f = M = 3.17 \text{ W}$ , and the actual fin heat transfer is overestimated by 17%.