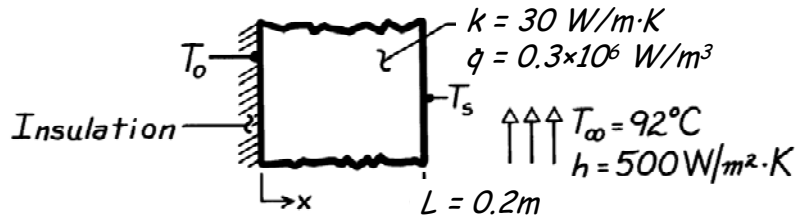


### PROBLEM 3.81

**KNOWN:** Plane wall with internal heat generation which is insulated at the inner surface and subjected to a convection process at the outer surface.

**FIND:** Maximum temperature in the wall.

**SCHEMATIC:**



**ASSUMPTIONS:** (1) Steady-state conditions, (2) One-dimensional conduction with uniform volumetric heat generation, (3) Inner surface is adiabatic.

**ANALYSIS:** The temperature at the inner surface is given by Eq. 3.48 and is the maximum temperature within the wall,

$$T_o = \dot{q}L^2 / 2k + T_s.$$

The outer surface temperature follows from Eq. 3.51,

$$T_s = T_\infty + \dot{q}L/h$$

$$T_s = 92^\circ\text{C} + 0.4 \times 10^6 \frac{\text{W}}{\text{m}^3} \times 0.2\text{m} / 500\text{W/m}^2 \cdot \text{K} = 92^\circ\text{C} + 160^\circ\text{C} = 252^\circ\text{C}.$$

It follows that

$$T_o = 0.4 \times 10^6 \text{W/m}^3 \times (0.2\text{m})^2 / 2 \times 30\text{W/m} \cdot \text{K} + 252^\circ\text{C}$$

$$T_o = 267^\circ\text{C} + 252^\circ\text{C} = 519^\circ\text{C}.$$

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**COMMENTS:** The heat flux leaving the wall can be determined from knowledge of  $h$ ,  $T_s$  and  $T_\infty$  using Newton's law of cooling.

$$q''_{\text{conv}} = h(T_s - T_\infty) = 500\text{W/m}^2 \cdot \text{K} (252 - 92)^\circ\text{C} = 80\text{kW/m}^2.$$

This same result can be determined from an energy balance on the entire wall, which has the form

$$\dot{E}_g - \dot{E}_{\text{out}} = 0$$

where

$$\dot{E}_g = \dot{q}AL \quad \text{and} \quad \dot{E}_{\text{out}} = q''_{\text{conv}} \cdot A.$$

Hence,

$$q''_{\text{conv}} = \dot{q}L = 0.4 \times 10^6 \text{W/m}^3 \times 0.2\text{m} = 80\text{kW/m}^2.$$