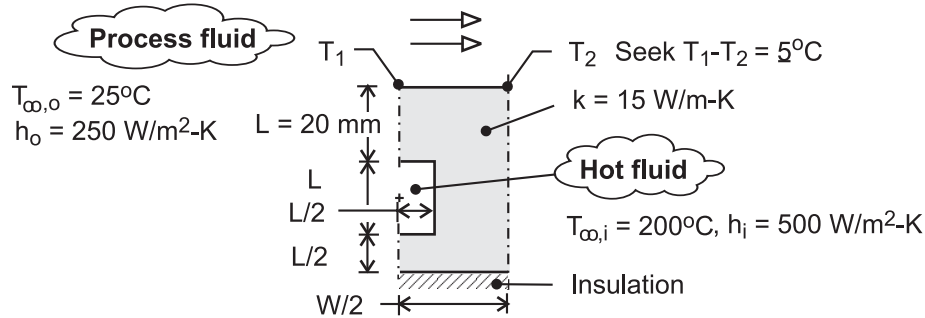


## PROBLEM 4.95

**KNOWN:** Upper surface of a platen heated by hot fluid through the flow channels is used to heat a process fluid.

**FIND:** (a) The maximum allowable spacing,  $W$ , between channel centerlines that will provide a uniform temperature requirement of  $5^\circ\text{C}$  on the upper surface of the platen, and (b) Heat rate per unit length from the flow channel for this condition.

**SCHEMATIC:**



**ASSUMPTIONS:** (1) Steady-state, two-dimensional conduction with constant properties, and (2) Lower surface of platen is adiabatic.

**ANALYSIS:** As shown in the schematic above for a symmetrical section of the platen-flow channel arrangement, the temperature uniformity requirement will be met when  $T_1 - T_2 = 5^\circ\text{C}$ . The maximum temperature,  $T_1$ , will occur directly over the flow channel centerline, while the minimum surface temperature,  $T_2$ , will occur at the mid-span between channel centerlines.

We chose to use FEHT to obtain the temperature distribution and heat rate for guessed values of the channel centerline spacing,  $W$ . The following method of solution was used: (1) Make an initial guess value for  $W$ ; try  $W = 100$  mm, (2) Draw an outline of the symmetrical section, and assign properties and boundary conditions, (3) Make a copy of this file so that in your second trial, you can use the *Draw | Move Node* option to modify the section width,  $W/2$ , larger or smaller, (4) Draw element lines within the outline to create triangular elements, (5) Use the *Draw | Reduce Mesh* command to generate a suitably fine mesh, then solve for the temperature distribution, (6) Use the *View | Temperatures* command to determine the temperatures  $T_1$  and  $T_2$ , (7) If,  $T_1 - T_2 \approx 5^\circ\text{C}$ , use the *View | Heat Flows* command to find the heat rate, otherwise, change the width of the section outline and repeat the analysis. The results of our three trials are tabulated below.

Trial	$W$ (mm)	$T_1$ ( $^\circ\text{C}$ )	$T_2$ ( $^\circ\text{C}$ )	$T_1 - T_2$ ( $^\circ\text{C}$ )	$q'$ (W/m)
1	100	108	98	10	--
2	60	119	118	1	--
3	80	113	108	5	1706

**COMMENTS:** (1) In addition to the tutorial example in the FEHT User's Manual, the solved models for Examples 4.3 and 4.4 of the Text are useful for developing skills in using this problem-solving tool.

(2) An alternative numerical method of solution would be to create a nodal network, generate the finite-difference equations and solve for the temperature distribution and the heat rate. The FDEs should allow for a non-square grid,  $\Delta x \neq \Delta y$ , so that different values for  $W/2$  can be accommodated by changing the value of  $\Delta x$ . Even using the IHT tool for building FDEs (*Tools | Finite-Difference Equations | Steady-State*) this method of solution is very labor intensive because of the large number of nodes required for obtaining good estimates.